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HOLLY SUGAR

SPECIFICATIONS

FOR

90" DIA. LIME SLAKER

HOLLY SUGAR CORPORATION

BRAWLEY FACTORY

GENERAL: These specifications describe a Rotary Lime Slaker in strict conformance with HOLLY SUGAR CORPORATION Inquiry dated March 16, 1983 (Requisition No. BY-1007-ED) and Holly Drawing No. A-90351-6.

The Slaker consists of a 90" diameter horizontal shell driven by a ring gear and rotating on steel tires supported by idler trunnions. Arrangement will be as shown by LUCAS ENGINEERING Drawing No. 72-953 enclosed.

OPERATING PERFORMANCE: The equipment proposed is designed to produce high quality Milk of Lime (M.O.L.) from normal calcined Lime Rock, without water burning or drowning. The internal agitating flights are interrupted to promote vigorous liquid circulation and set at a sufficiently low helix angle to prevent lifting of the solids above the liquid level.

Internal flights are arranged in four rows 90° apart, so that at least one row is immersed and agitating the M.O.L. at all times.

CONCURRENT OPERATION

COUNTER-CURRENT OPERATION

Milk of Lime:

Apparent Density	32° Bx	32° Bx
CaO, in % on Beets	3.6%	3.6%
Production Rate	280 gpm	280 gpm
Retention Time	2.6 min	2.6 min
Outlet Temp.	180° F	212° F
Reynolds No.	6.9×10^7	7.3×10^7
	vs	vs
N _{Re} reqd for Turbulence	1×10^4	1×10^4

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	<u>CONCURRENT OPERATION</u>	<u>COUNTER-CURRENT OPERATION</u>
Calcined Lime:		
Production Rate	325#/min	325#/min
Bulk Density (dry)	50#/cu. ft.	50#/cu. ft.
" " (hydrated)	33#/cu. ft.	33#/cu. ft.
Volumetric Rate	10 cu. ft/min	10 cu. ft/min
Inlet Solids/Liquid Ratio	1:4	1:4
Design Conveying Rate	3.0 ft/min	3.0 ft/min
Residence Time	10 min	10 min
Dross, % on Rock	5%	5%

MECHANICAL DESIGN: The Slaker and all mechanical components have been conservatively designed to provide indefinite life. Individual components are as follows:

Rotating Shell.....3/8" Thk A-36 Plate
 Tires.....100" O.D. x 94" ID x
 6" Face Width
 Idler Trunnions.....13½" O.D. x
 7" Face Width
 Trunnion Shaft.....5" dia x 4" @ Brgs
 Trunnion Bearings.....Timken Type "AP"
 Class B (4¼ x 8)
 Ring Gear (AGMA Class 6).....2 D.P. x 200 Teeth
 x 5" Face Width
 Pinion (AGMA Class 6).....2 D.P. x 30 Teeth
 x 5½" Face Width
 Thrust Rolls.....9" dia x 3" Face Width

Mechanical Design parameters are:

	<u>Actual</u>	<u>Allowable</u>
Tire/Trunnion Loading	7275 lbs vs	20250 lbs
Trunnion Bearings, L ₁₀ Rated Life	32,500,000 hrs	N/A
Trunnion Shaft Stress	1371 psi	25,000 psi
Pinion: Tooth Bending	12,154 psi	22,000 psi
Surface Durability	9430 psi	50-60,000

The ring gear is mounted to the shell by means of double wedges which will be adjusted in the shop to 1/16" T.I.R. (+1/32") and welded in position. The adjacent tire is located concentric with the gear by a machined pilot, so that pitch line contact of the gear and pinion is virtually independent of runout.

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The pinion is arranged to dip into a reservoir of "Extreme Pressure" lubricant, providing continuous lubrication for itself and the ring gear.

Continuous lubrication of the tires and trunnions is not considered necessary, especially in the dirty lime kiln area, although it can easily be provided at the customer's option.

DRIVE: The proposed Slaker is driven by the pinion mounted directly on the low-speed output shaft of the speed reducer, driven by a V-Belt Drive from an electric motor.

Drive components are:

Speed Reducer:

Manufacturer.....Hansen
Type.....JFN Concentric Shaft
Size.....81C-Double Reduction
Ratio (Actual).....25.0:1
Service Factor.....2.6
Low Speed Shaft:
O.H. Load.....13,400 lbs. (Allowable)
" " 5,408 " (Actual)

V-Belt Drive:

Manufacturer.....Dodge
Type.....Dyna-V
Belts.....Four-3V850
Driven Sheave.....10.6 O.D. w/2" Taper-Lock Hub
Driving Sheave.....5.6" O.D. w/1 5/8" Taper-Lock Hub

Motor:

Frame 254 T
Size.....15 H.P.
Type.....Horizontal
Enclosure.....TEFC
Elec. Characteristics.....460V3ph 60 Hz

PAINTING: Exterior surfaces, not machined, will be painted one coat shop primer. Machined surfaces will be given one coat Rustoleum rust preventive compound.

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SHIPPING AND ERECTION Shipment will be made by contract motor freight carrier
for delivery to installation site.

WEIGHTS:

Calculated Shipping Weights are:

Shell Assy with Tires and Ring Gear.....	17,800 lbs.
Trunnion Cradle, Drive End.....	2,900
Trunnion Cradle, Idler End.....	1,500
Speed Reducer.....	680
Motor.....	162
V-Belt Drive, w/Guard.....	130

FIELD The shell will be shipped with ring gear and tires mounted true to
ERECTION: T.I.R. ($\pm 1/32"$). No further field adjustment will be necessary.

Trunnion cradles will be shipped with trunnion roll faces in same plane and axes parallel. However, because field mounting on foundations will inevitably involve slight misalignment between the two cradles, some adjustment of trunnion alignment will probably be required. Jackscrews for this adjustment are provided for each trunnion bearing.

The following sequence of installation is recommended:

1. After setting slaker on trunnions, adjust thrust rollers to within $1/8"$ of thrust ring faces ($1/4"$ total float).
2. Check bearing of trunnion faces against tires with feeler gauges and adjust as necessary.
3. Install speed reducer and electric motor drive on shovel mount provided. Bring pinion into engagement with ring gear and adjust to .065 backlash for preliminary tooth bearing.
4. Fill pinion reservoir with E.P. lubricant.
5. Set up dial indicator and jog ring gear one or more revolutions to locate maximum run-out, and bring that point to conjunction with pinion.
6. Increase engagement to .045 minimum backlash across teeth.
7. Rotate shell several revolutions, observing lubricant pattern for uniform bearing across tooth face and skew reducer as necessary to obtain same.

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8. During step No. 7 also observe behavior of tire against thrust rollers. Correct behavior is for slaker to float between the thrust rollers, never in contact for more than a fraction of a revolution of the slaker.
9. To correct one-directional float, apply lubricant to both tires and observe pattern following trunnion contact. Back of one bearing on each cradle to obtain uniform pattern (advancing other bearing is not recommended as it may reduce backlash to unacceptable minimums).
10. Securely lock all adjustment and observe at frequent intervals for proper operation during first 24 hours of operation, and once each shift thereafter.
11. Note: Shell will expand about $\frac{1}{4}$ " from 70° F ambient to operating temperature. Tires should be centered on trunnions when operating at normal rate.

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